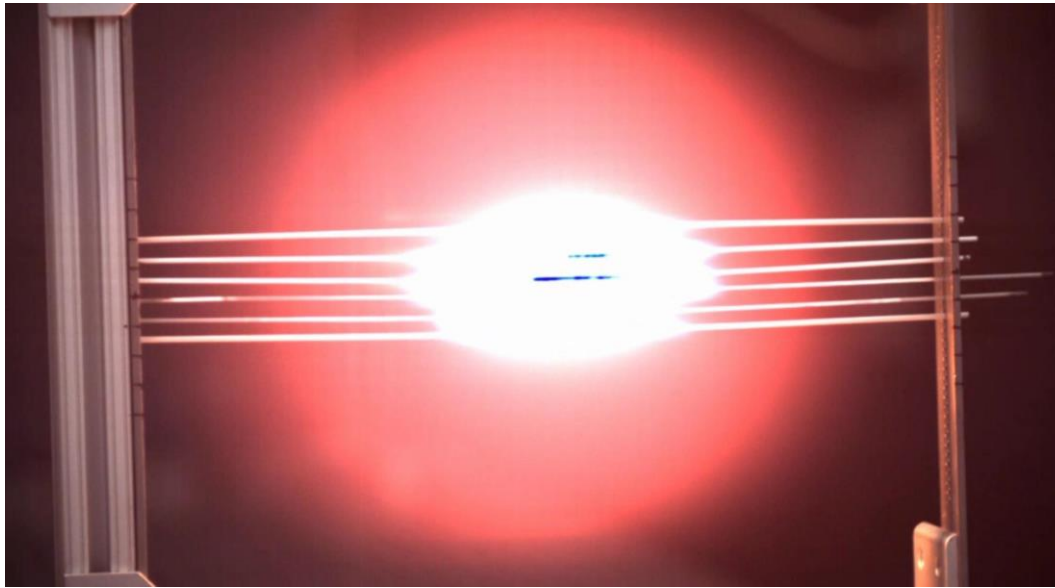


Development and Testing of a Refractory Millimeter-Wave Absorbent Heat Exchanger

Thomas Lambot^a, Leik Myrabo^b, David Murakami^c, Kevin Parkin^a



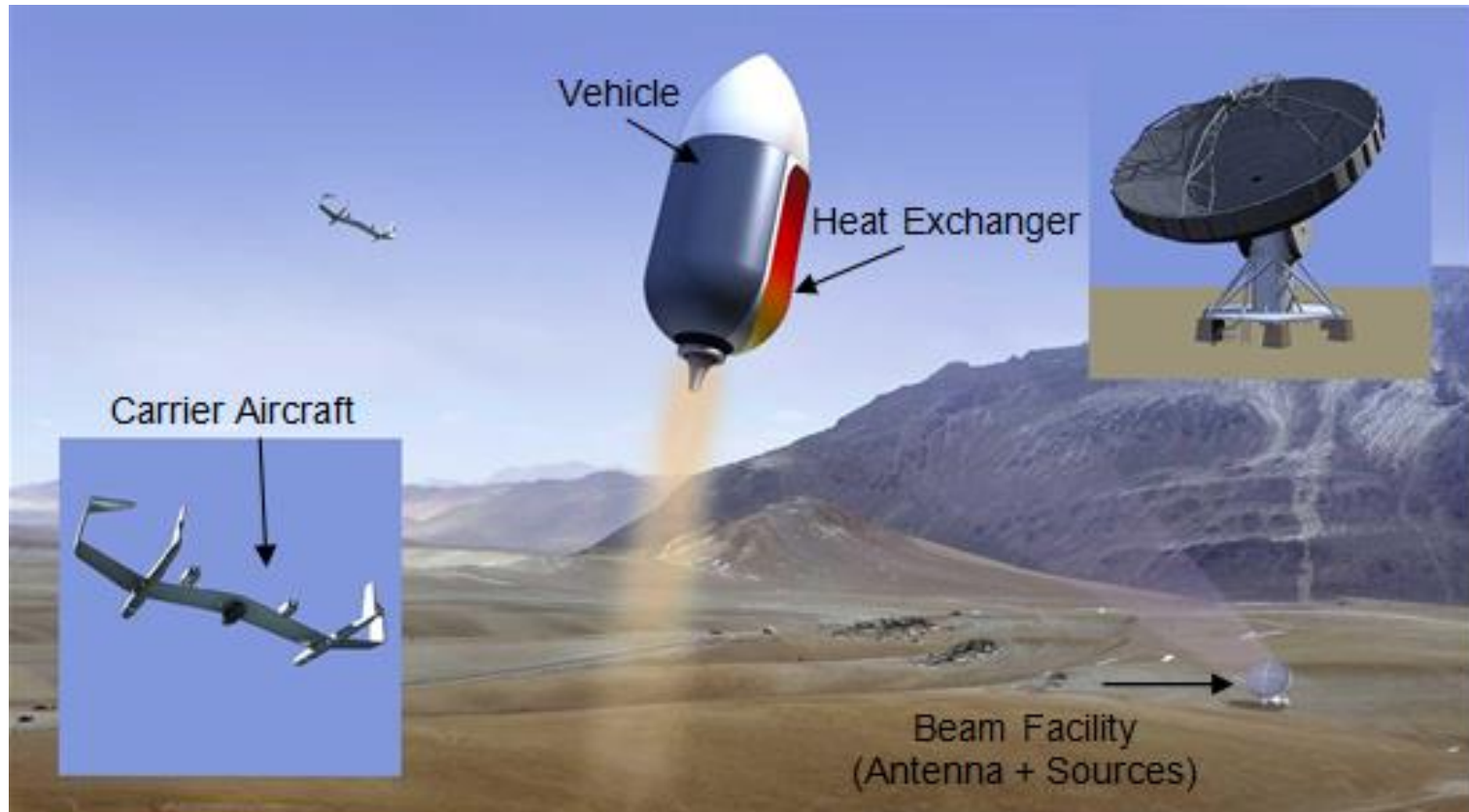
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^c NASA Ames Research Center, Moffett Field, 94035, United States of America

"The views expressed are those of the author and do not reflect the official policy or position of the Department of Defense or the U.S. Government."

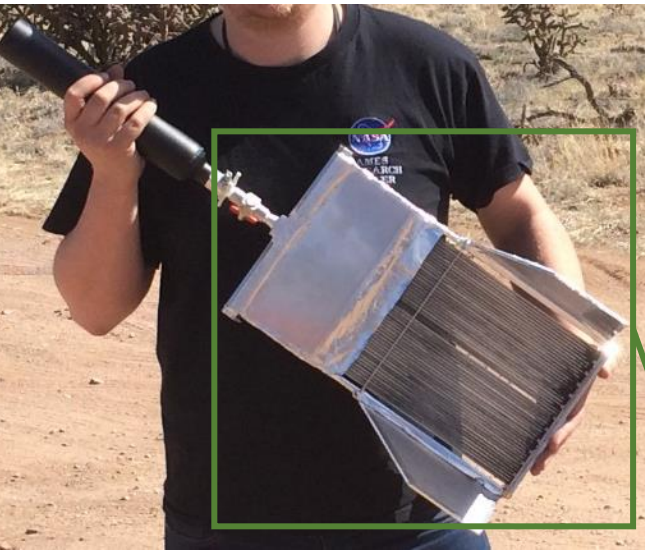
The Millimeter-wave Thermal Launch System (MTLS)



Artist view of the MTLS concept

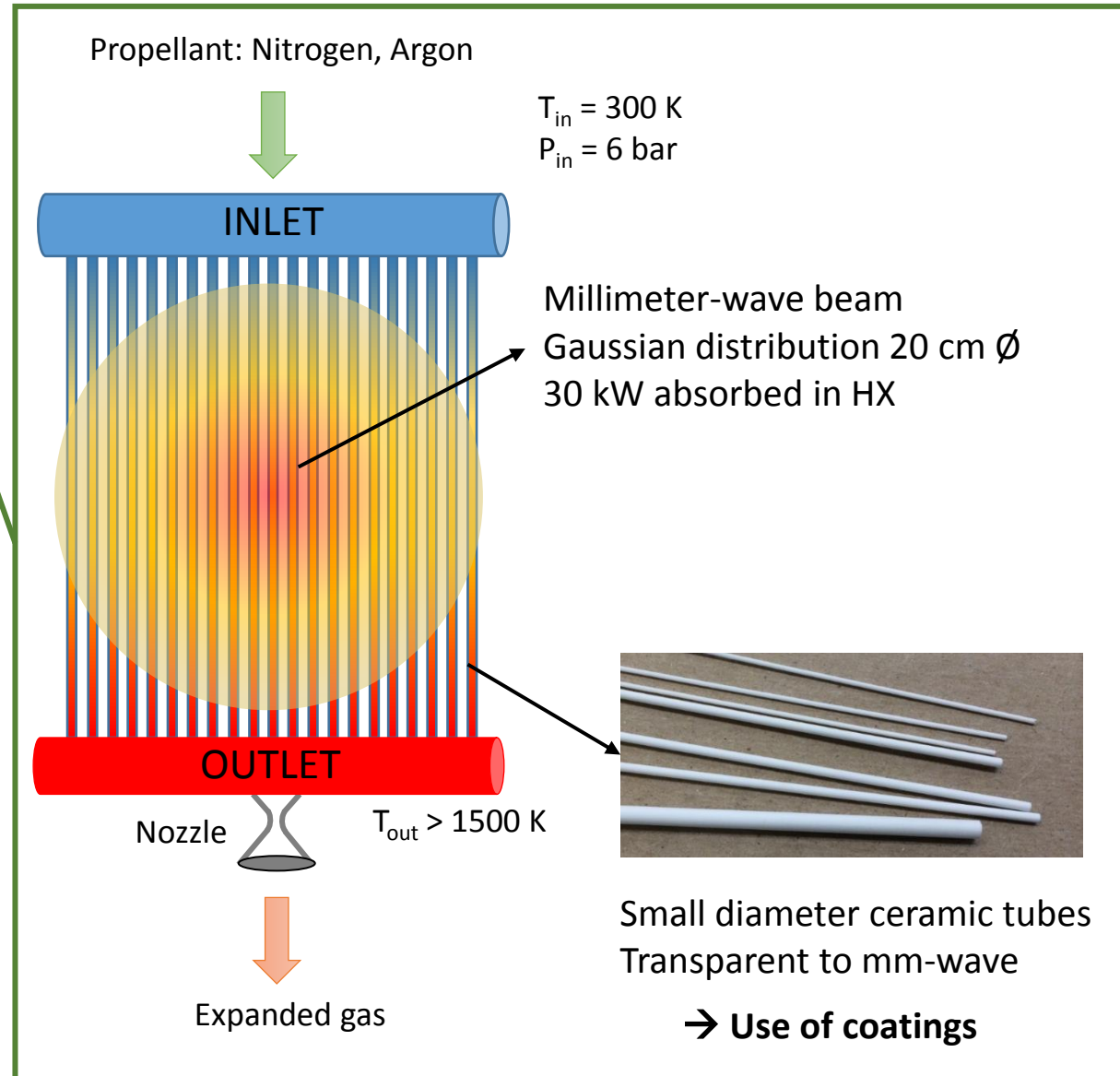
Thermal propulsion using beamed energy as the source of power

The planar heat exchanger: point of design



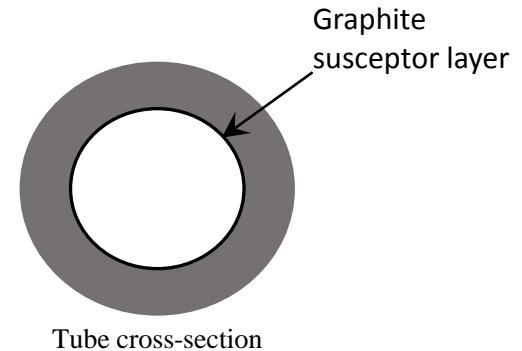
Requirements:

- Absorb the mm-wave beam
- Transfer heat efficiently to propellant
- Withstand high temperature (> 1500 K)
- Lightweight



Solution: Internal graphite coating

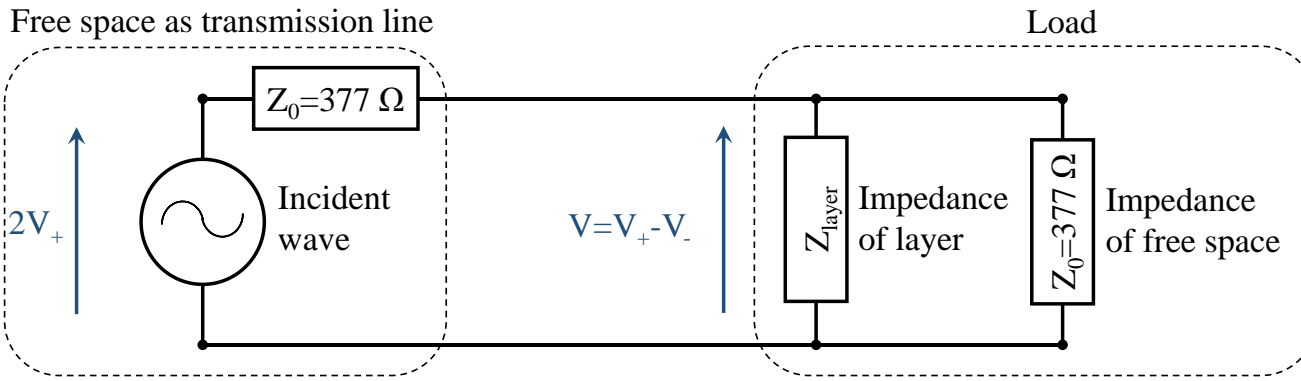
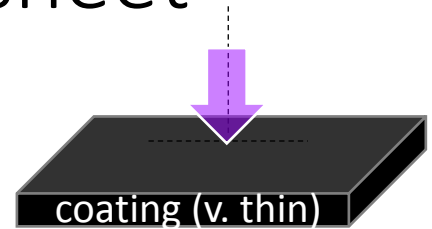
- Coating on the internal part of the tube
 - Non-oxidizing atmosphere
 - Heat source is next to the flow
- Use of “Aquadag”: A resistance paint
 - Composed of colloidal graphite in solution with water
 - Ohms per square of the coating are tuned by adjusting the mixture ratio
- Tubes are coated by using a pump system that flows the coating through the tube
 - Cured in a vacuum oven for 3 hours
 - Multiple coatings give best uniformity



Absorption by a thin conducting sheet

- Thévenin equivalent circuit

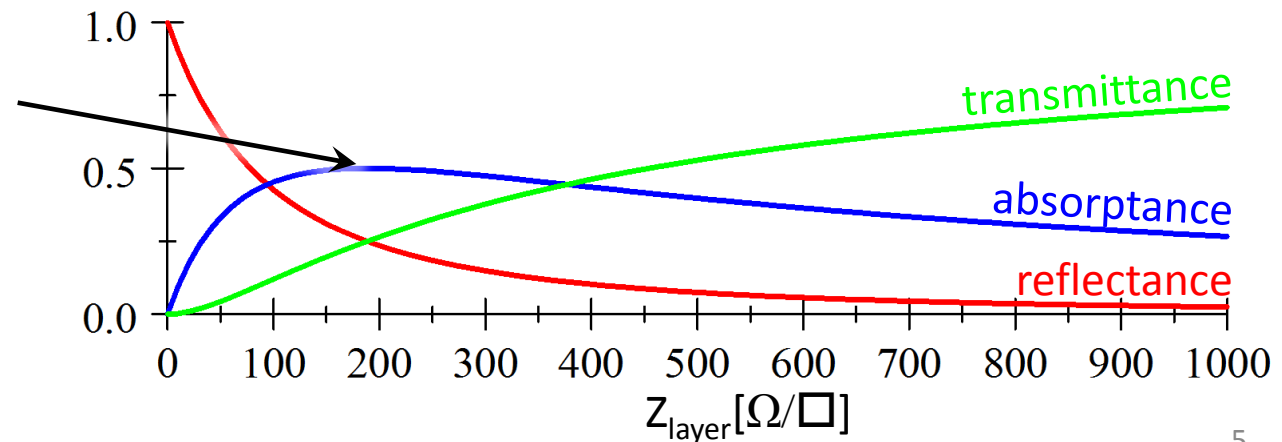
- Transmission line model of reflection for planar conducting sheet in free space illuminated at normal incidence by an incident wave



- Based on this circuit, and defining $r = \frac{Z_{layer}}{Z_0}$, it can be shown that:

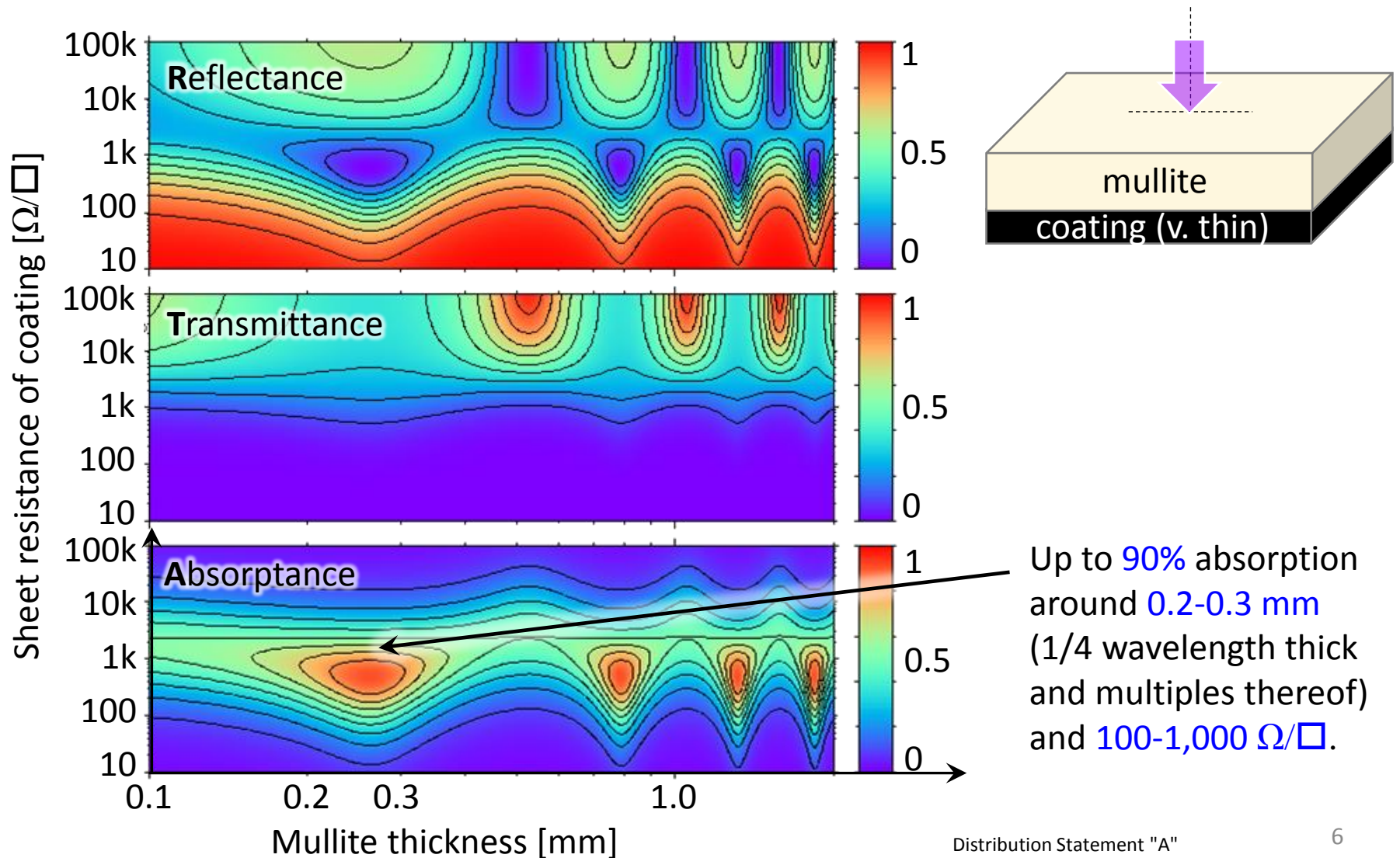
- Absorptance $A = \frac{r}{(r + \frac{1}{2})^2}$, Reflectance $R = \frac{1}{(1 + 2r)^2}$, Transmittance $T = \frac{r^2}{(r + \frac{1}{2})^2}$

A maximum of 50% of the incident power can be absorbed by a sheet on its own



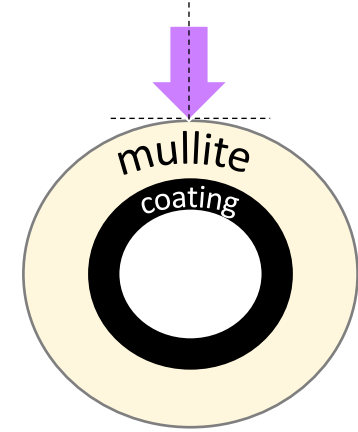
Planar stratified layer

- The stratified layer model can include the mullite 'anti-reflection' layer. Predictions for normal incidence are shown here.

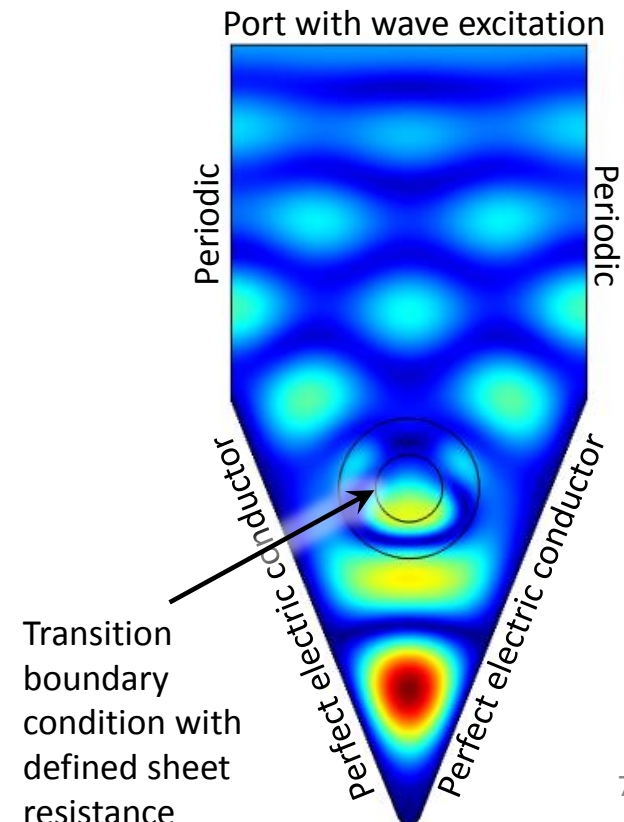


Finite element

- In reality, the tube is curved
- The tube is at wavelength scale
- The tube is not isolated, but in an array
- Can a back-reflector be contoured to improve absorption, uniformity of heating around the tube (to reduce transient bending)?
- **Work is still underway in COMSOL to address these questions**
- We want to produce performance maps, similar to those just shown for the stratified layer model, and compare them to experimental results.
- The example shows norm of electric field for wave incident on tube with interior coating and V shaped back-reflector



Tube cross-section



Coating experiments at General Atomics DIII-D facility

- Test campaigns at General Atomics in our Mobile Test Unit (MTU) to investigate absorptivity of materials
 - Use of 110 Ghz gyrotron at DIII-D facility
 - Test of material coupons, coated tubes, concentrating mirrors, ...

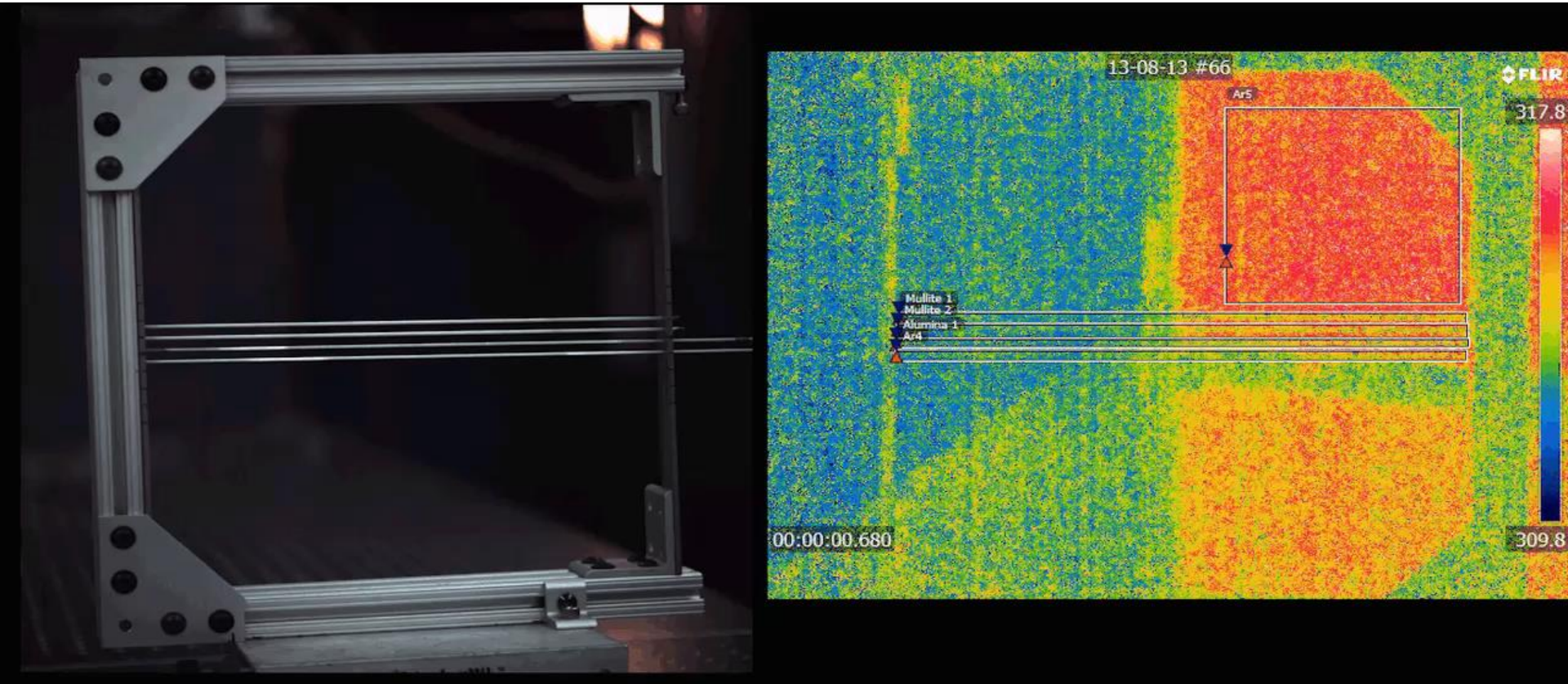


Initially: tubes beamed for 50 to 1000 ms at peak power intensity of $1\text{-}2\text{ kW/cm}^2$

After refinement: tubes beamed up to 3s at $250\text{ to }530\text{ W/cm}^2$



Example of tube testing



Mullite and Alumina tubes with internal Aquadag coating (mixture ratio 1:1)
Maximum temperature reached: 1650 K



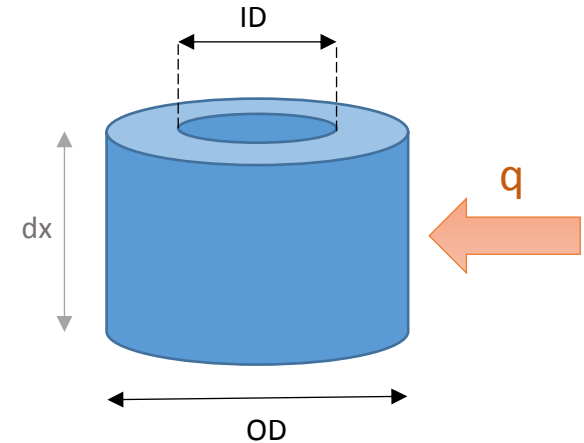
Tube broken due to non uniformity of coating

Absorption results

- Consider the beaming of a tube section dx with an inner diameter ID and outer diameter OD
- Simplifications:
 - Beam absorbed uniformly throughout the mass
 - Section at uniform temperature $T = T_{max}$
 - Assume power density q from the incident beam is uniform on the section
 - Graphite layer weight and specific heat capacity are neglected

Heating rate if the tube was a black body:

$$\left. \frac{dT}{dt} \right|_{black\ body} = 4 \frac{q}{c_p(T) \rho \pi (OD^2 - ID^2)}$$

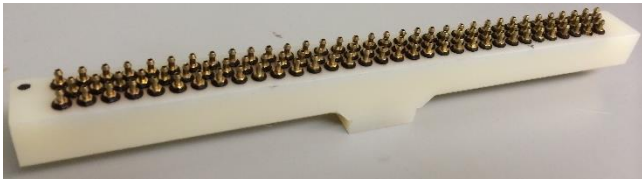


$$A = \frac{dT/dt|_{measured}}{dT/dt|_{black\ body}}$$

- Campaign results (preliminary, work still in progress!)

Tube type		Absorptance A [-]
Alumina 0.794 mm ID, 0.39 mm wall	Uncoated	0.02
	Coated with 1:1 mix ratio	0.22 ... 0.28
Mullite 0.794 mm ID, 0.39 mm wall	Uncoated	0.08
	Coated with 1:1 mix ratio	0.25 ... 0.5 (?)
Mullite 1.19 mm ID, 0.395 mm wall	Uncoated	0.05
	Coated with 1:1 mix ratio	0.45 ... 0.58

The heat exchanger assembly



Nylon inlet manifold with barbed fittings



Flow-restricting orifice

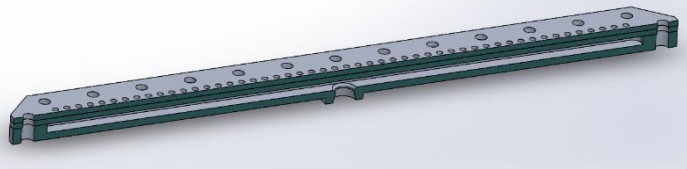


Polyurethane tubing with crimped clamp



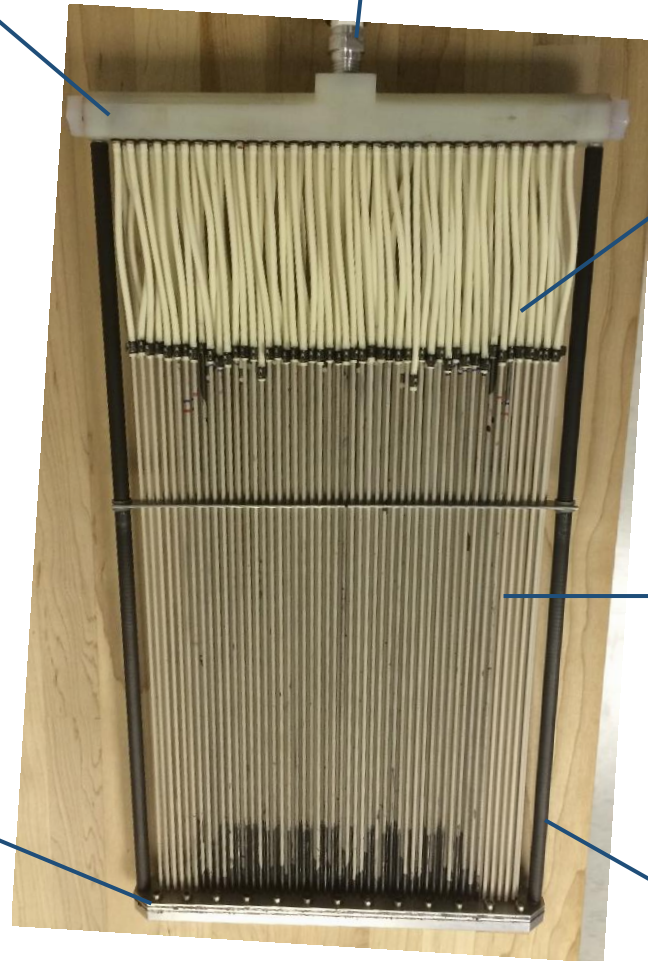
Flanged titanium outlet with graphite gasket

Carved nozzle: $\phi_t = 7.6\text{cm}$ and $\frac{A_E}{A_t} = 1.5$



100 Mullite tubes
1.19 mm ID, 0.395 mm wall
Coated with Aquadag

Carbon fiber tube (strut)

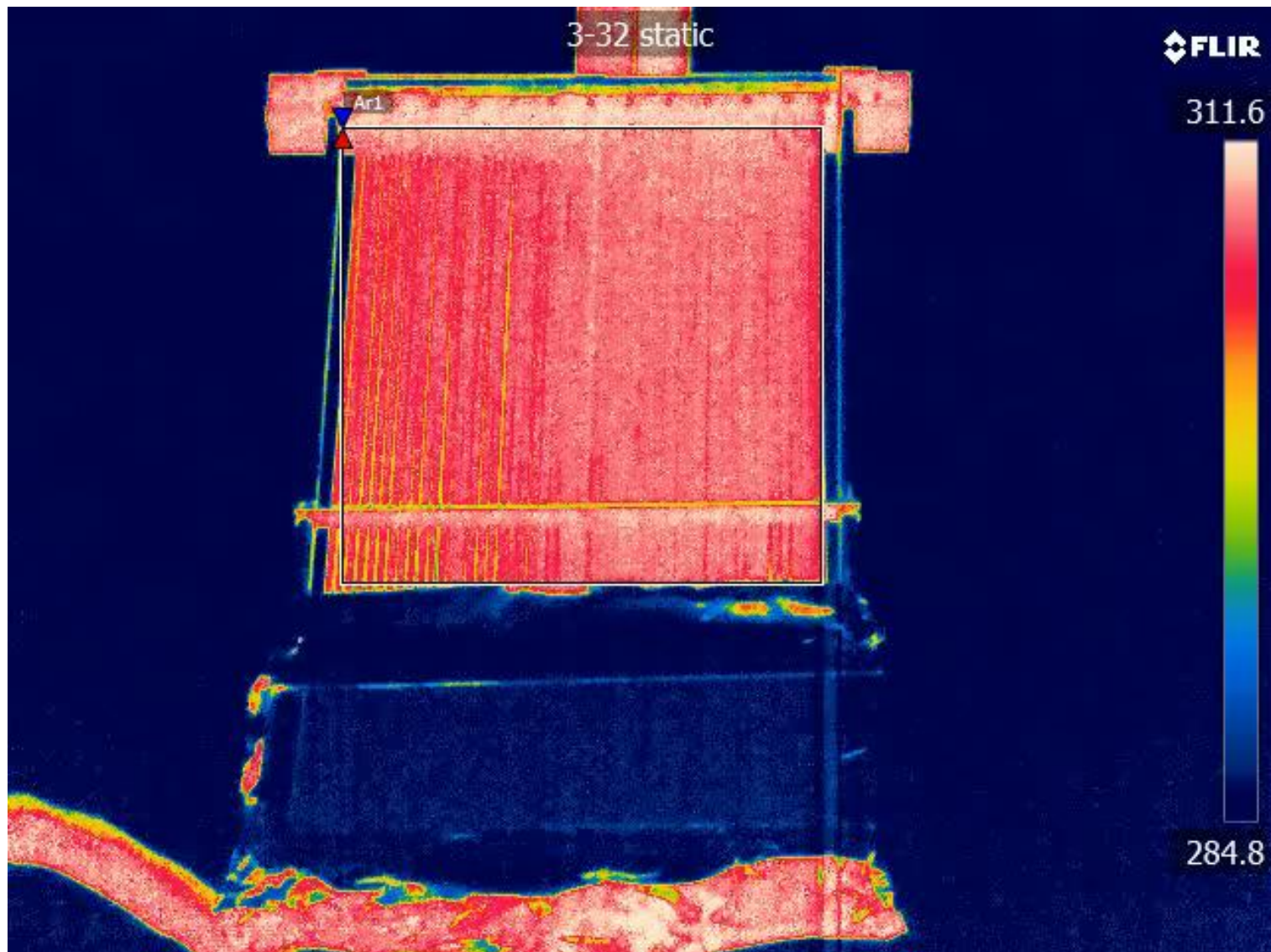


Test Arrangement



- Testing using 100 kW-class W-band source
 - Gaussian-like beam focused onto heat exchanger
 - Point and tracking system off for static test
 - Pre-heating of heat exchanger using the beam
 - **6 to 10 s** total shot duration
 - Use of **Argon** for propellant
- Shot parameters
 - Power peak intensity of **71 W/cm²**
 - **33 kW** in a 30 cm Ø bucket

Static test using a 3-32" orifice with Argon

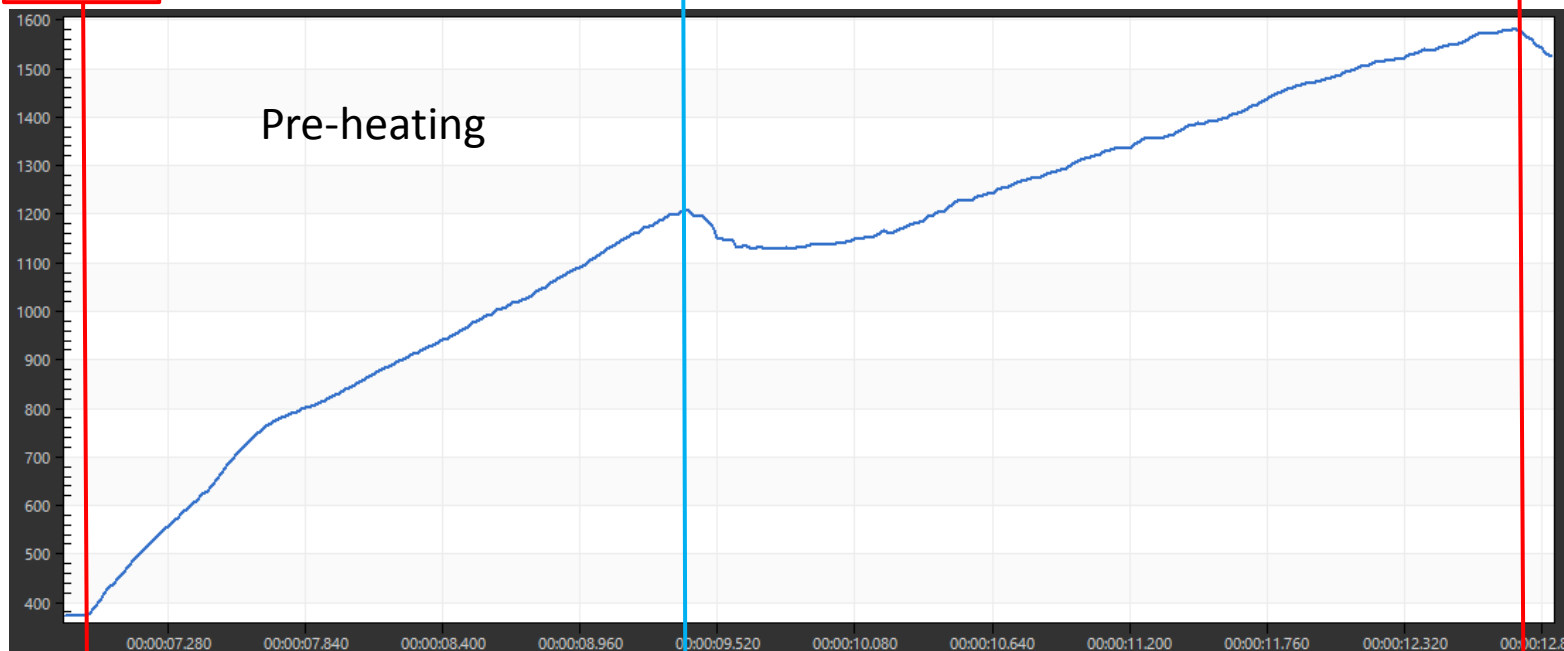


Heat exchanger
wall max T
[K]

Beam on

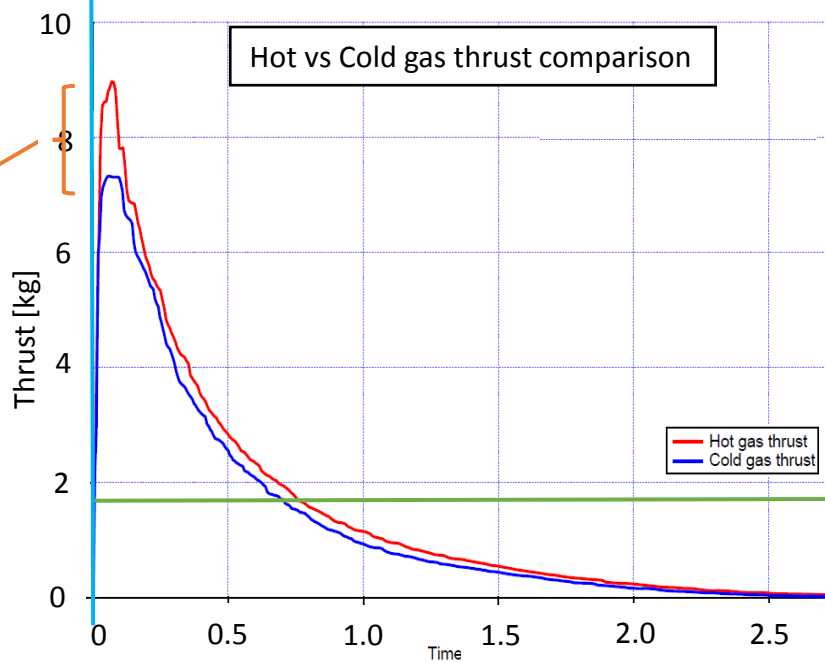
Flow on

Beam off

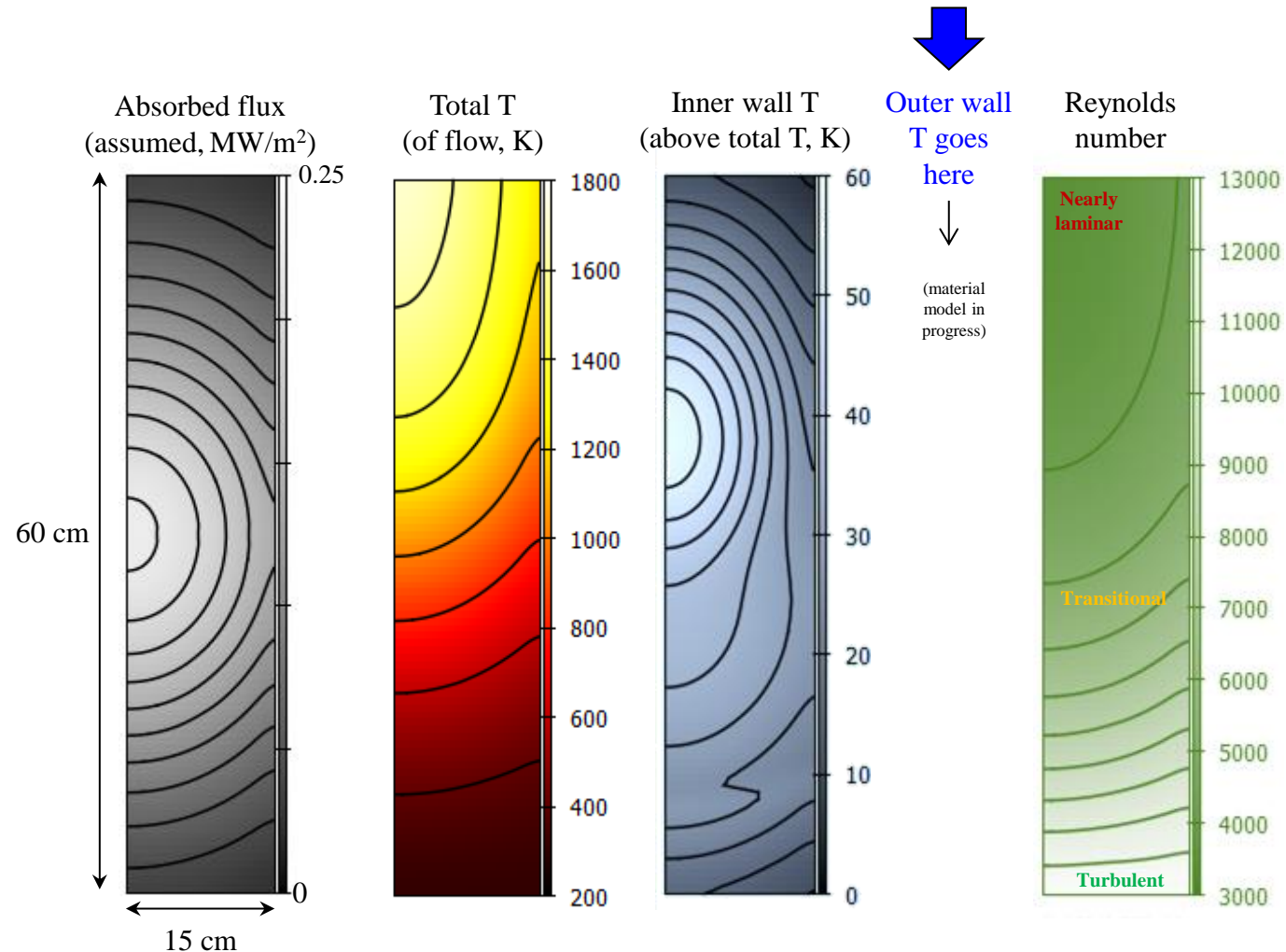
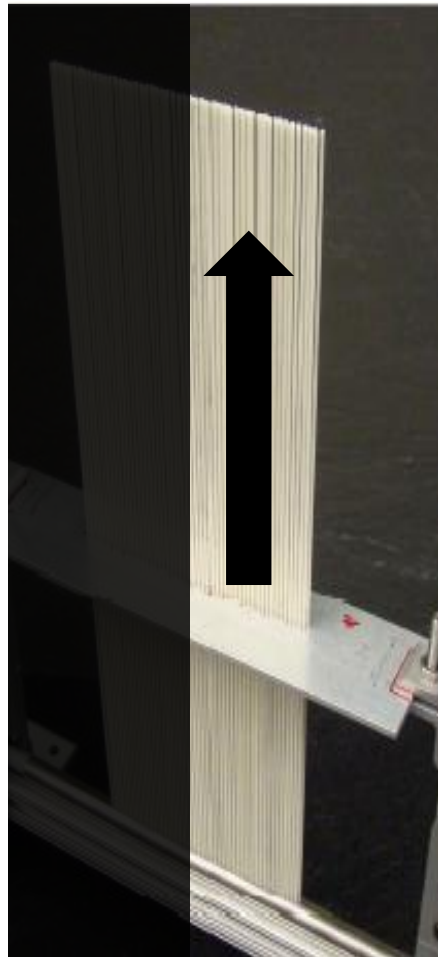


25 % increase in thrust
when comparing test with and
without the beam on
→ Heat transfer to investigate ...

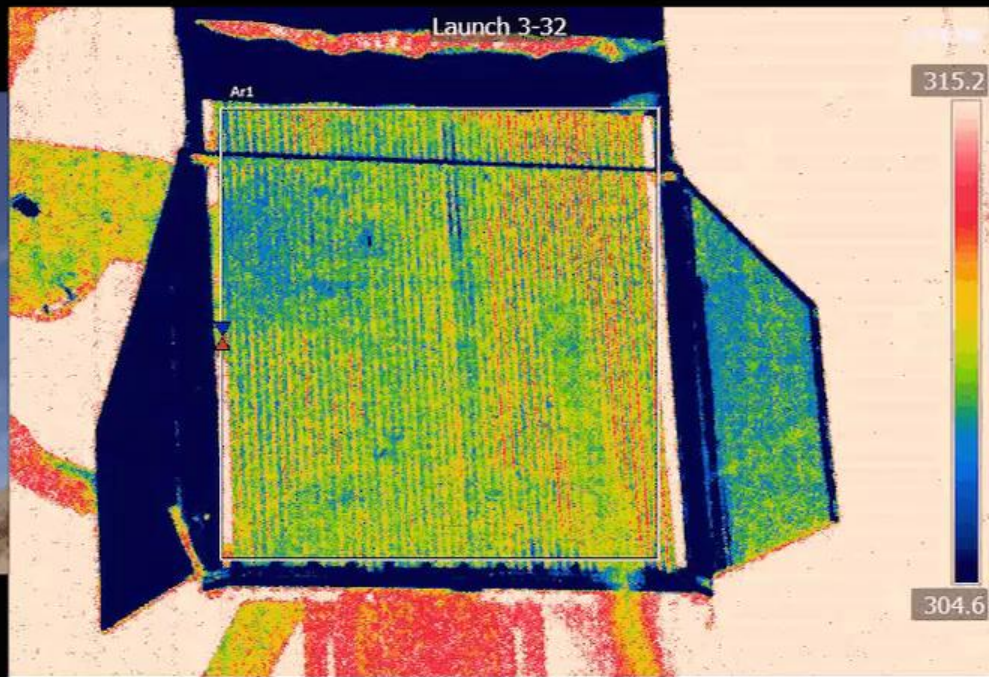
Steady state not reached as
mass flow rate is not constant



Quasi-1D simulations



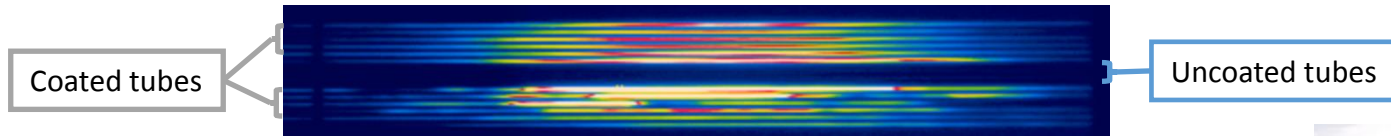
First flight of MM-wave rocket



Highlights & Future work

- Achievements

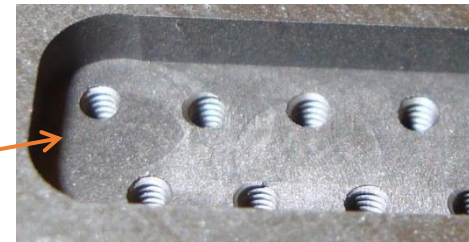
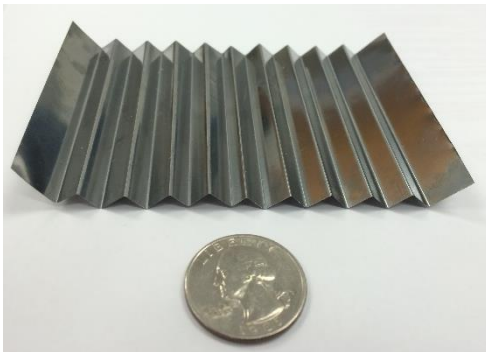
- Using a graphite coating, we turned mm-wave transparent ceramic into an absorber



- Tube surface temperatures higher than 1700 K (1500 K with flow of helium) were reached without damaging tubes
- Heat exchanger prototypes were assembled, static tested and launched

- Future optimization

- Improve coating quality (absorptance and uniformity)
- Improve convective heat transfer and absorption efficiency
- Conduct finite element absorption simulation
- Investigate foil backing concentrator, threaded tubes, other materials for higher temperatures, ...



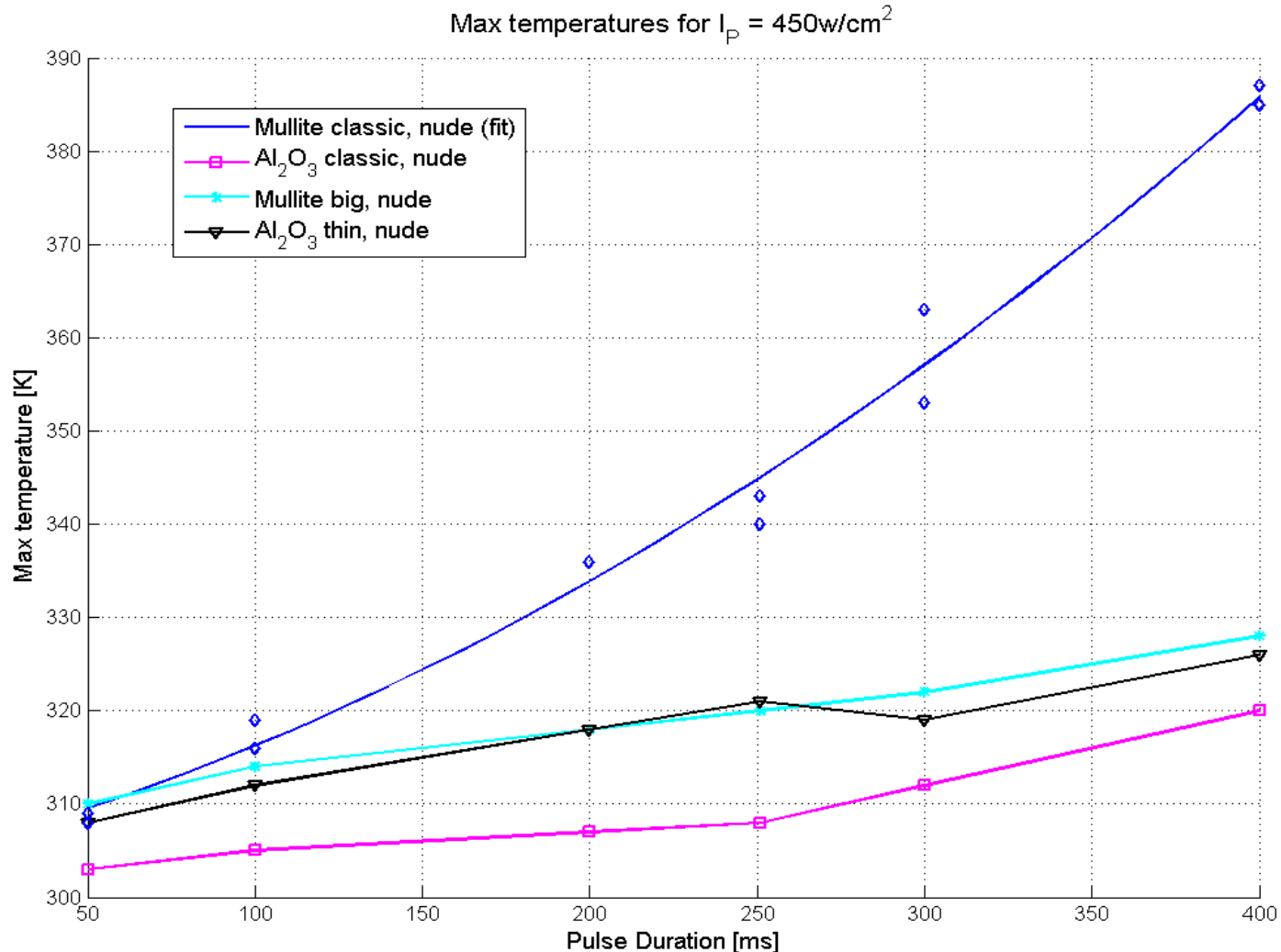
Questions



Distribution Statement "A"
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Backup slides

Bare tube absorption



“Mullite classic” absorbs better than the other tubes: This behavior is expected as it is composed of 80% mullite & 20% glass (glass is an absorber).

Overall, the tubes’ absorption on their own is negligible.

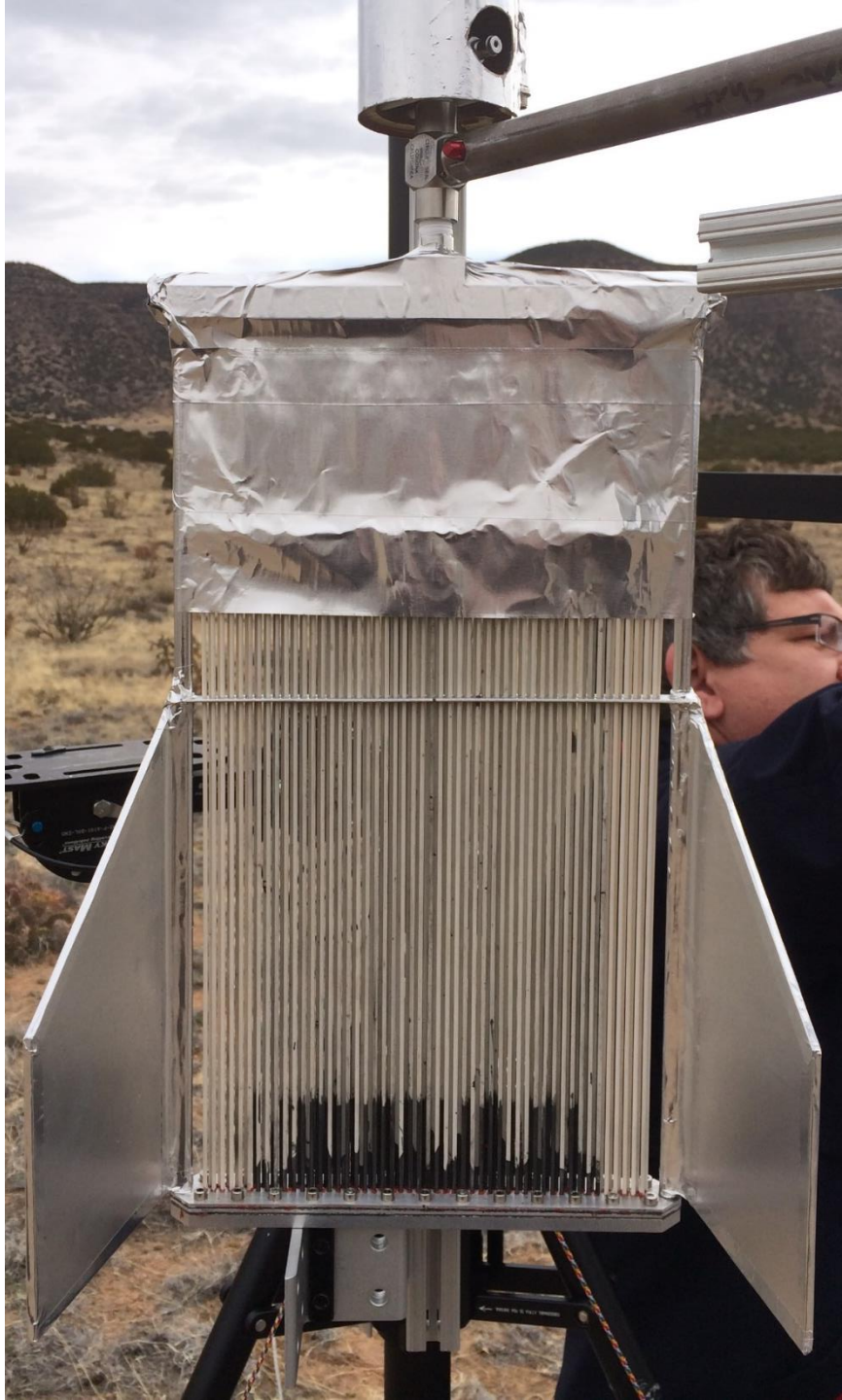
Non-uniform HX illumination

All channels same diameter

Optimally tapered channel diameters

	Fixed Channel Diameter	Variable Channel Diameter
Outlet $\langle T_t \rangle$ (K)	1,521	1,700
I_{sp} (sec)	91	97
Thrust (N)	21	19
\dot{m} (g/s)	23	20
No. of channels	191	233
Mass of channels (g)	670	570
Areal density (kg/m ²)	3.7	3.2

Decision: Fixed channel diameter good enough for now. Variable channel diameter model deferred.



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